

Curved Tooth Couplings
Basic Series for Industrial
and Marine Techniques



Curved Tooth Couplings

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Curved tooth couplings

from RENK, drive elements with profile and tradition. More than 60 years of experience in development and production of these many-sided shaft connections mean competence. Competence in drive applications. This is testified by the most extensive supply of gear couplings all over the world. The comprehensive range of curved tooth couplings extends from the torque giant with a bore diameter of 1200 mm to the Turbo design for speeds up to 40000 rpm. Standard types are available in common sizes directly from stock. Computer-optimized designs and the latest production and testing equipment ensure that our products always reflect the state-of-the-art. The standard range has thus also been revised again.

Improvements for the user's benefit

- The nominal size of the coupling is no longer synonymous with the max. permissible hub bore.
- Larger permissible hub bores permit more favourable size selection.
- Increased values for permissible angular misalignment offer more safety for shaft misalignment.
- Differentiated size determination by application of operating factors.
- Simplified selection thanks to new, clear range structuring.

A range with method

Divided into three groups with different design features, the new range offers a comprehensive selection of curved tooth couplings of widely varying types. There are three groups because each application places its own special demands on a shaft con-

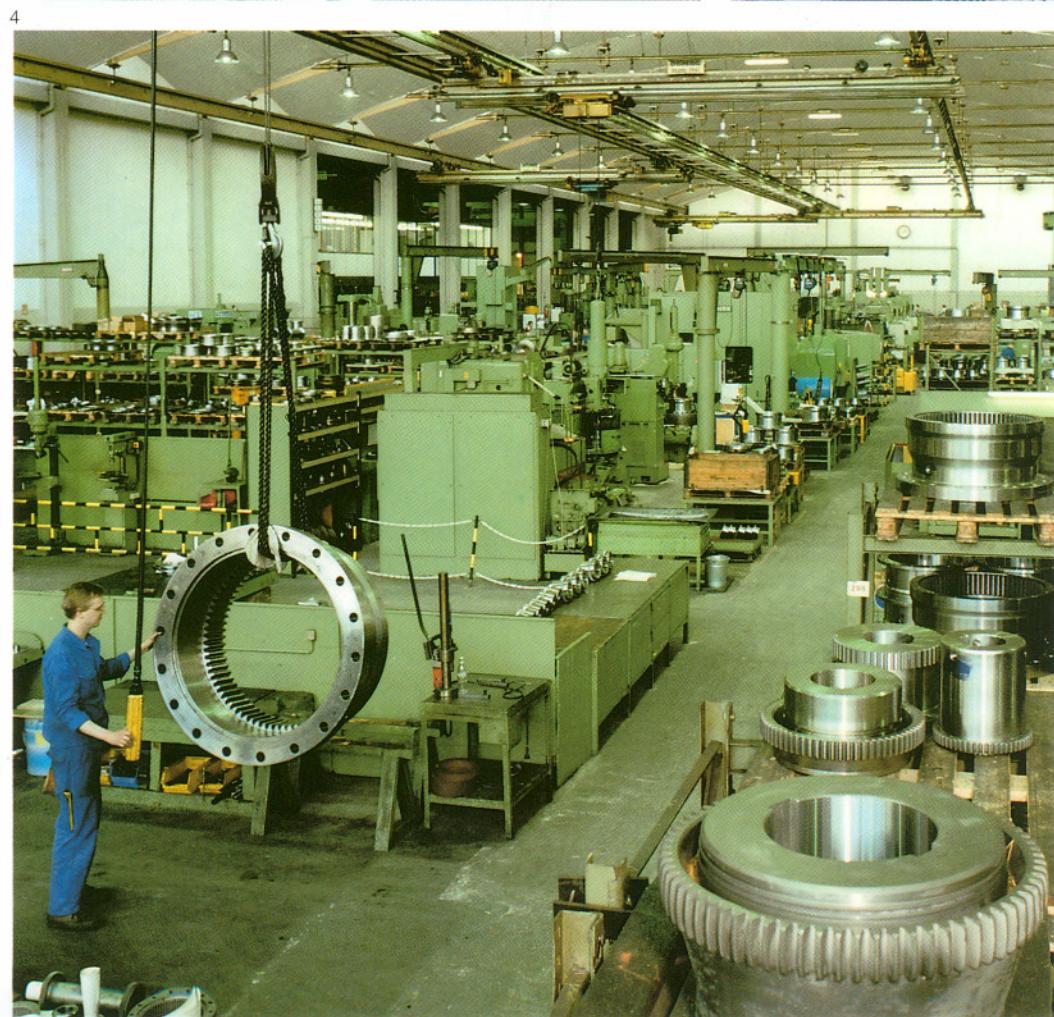
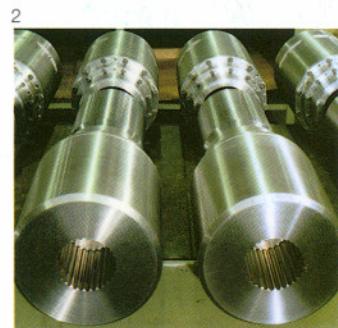
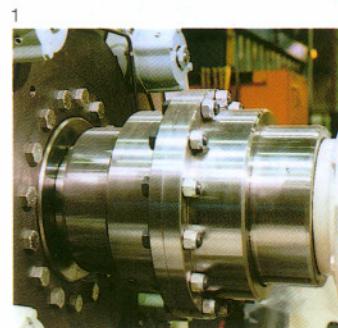
nexion. Some of these demands can be met with simple solutions,

but sometimes only complex de-

sign measures are the answer. The foremost objective is always to provide the best technical solution which also offers optimum economy.

Highest quality

goes without saying. The same applies equally to all designs. Design differences do not have any influence on the quality of the coupling or its individual parts. All components are produced in accordance with the same strict quality regulations.



1 Curved tooth coupling of the LBk series in a wind energy converter

2 Curved tooth spindle couplings for rolling mill drives

3 Curved tooth coupling of the SB series in the drive of a cement mill.

4 Partial view of the production hall for curved tooth couplings

Since 1989 the Renk Quality System has been implemented and certified in compliance with the DIN-ISO 9001 Standard.

Oil or grease lubrication

This choice exists for all couplings of the SB series.

An advantage of oil lubrication is the simpler and faster lubricant change. In addition, the SB types offer a larger lubricant chamber with protected contents. Even if the seal is damaged, lubrication is maintained. However, the same applies equally to both oil and grease lubrication.

Available ex stock

This is true for all basic designs in the coupling sizes up to 200 or 225. Depending on type series, this therefore applies to hub bores with a diameter of 12 mm up to max. 260 mm. The same is also valid for designs with spacers or intermediate shafts, but without these additional parts. Even couplings subject to acceptance by the classification societies GL, LRS or DNV are not excepted from this.

Delivery ex-stock means rapid help in the event of damage.

Special designs

are available in any required size, irrespective of quantity. A distinction must be made here between modified standard types and special designs. Modified designs essentially consist of elements of the basic series which have been slightly changed or equipped with additional components. Pure special designs, on the other hand, are normally complete new designs. The already produced examples on pages 38 to 41 show a selection of the extensive special range.

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5 Curved tooth couplings of a wide variety of types in the intermediate dispatch store

6 Curved tooth couplings of the RAG series from the Rafinex program

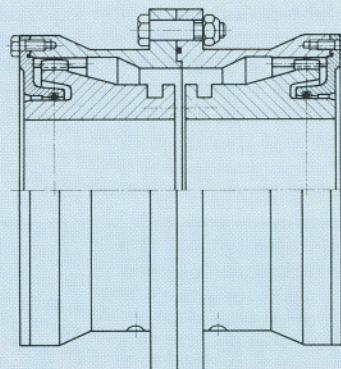
7 Final assembly of curved tooth couplings of the HBk series

8 Assembly of a curved tooth coupling of the size SBL 525 for a steel plant

Quality Characteristics

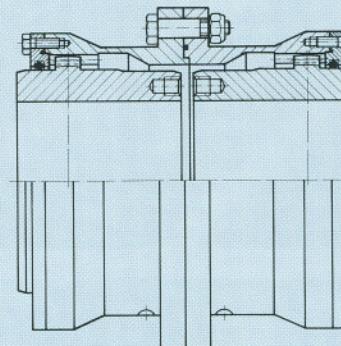
SB Series

- Technically complex design
- Split housing with bolted-on covers
- Elaborate cover design for high misalignment capacity
- Misalignment $\pm 1.5^\circ$ per coupling half, in special design up to $\pm 3^\circ$
- Lubrication optionally by oil or grease
- Low load on sealing rings thanks to optimum arrangement in housing cover
- Large lubricant chamber with protected contents
- Preservation of lubrication even in the event of damaged seal
- Easy replacement of sealing rings
- Tooth tip centering, optionally cover centering possible
- Large tooth center spacing
- Combination possibility with HYGUARD® safety elements, brake disks, torque measuring shafts or other attachments.



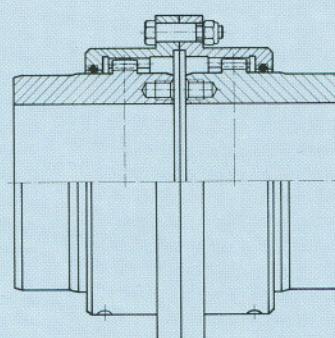
SBk Series

- Split housing with bolted-on covers
- Misalignment $\pm 0.75^\circ$ per coupling half
- Grease lubrication, optionally with long-time grease filling
- Easy replacement of sealing rings thanks to unscrewable cover
- Tooth tip centering
- Large tooth center spacing
- Large hub bores
- Interchangeable hubs to cope with different shaft spacings
- Combination possibility with HYGUARD® safety elements, brake disks, torque measuring shafts or other attachments.



LBk - Series

- Split housing with fixed covers
- Misalignment $\pm 0.75^\circ$ per coupling half
- Grease lubrication, optionally with long-time grease filling
- Tooth tip centering
- Large hub bores
- Low weight thanks to compact and light housing
- Variable overall length thanks to shortenable hubs
- Combination possibility with HYGUARD® safety elements, brake disks, torque measuring shafts or other attachments.



Summary Contents



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Coupling Selection and Size Determination

Table 1, Service Factor

Machine	Service Factor $K_1^{1)}$	Machine	Service Factor $K_1^{1)}$	Machine	Service Factor $K_1^{1)}$
Excavators		Wood processing machines		Presses	
Chain bucket excavators	2,0	Debarking drums	1,8	Folding presses	1,8
Travelling gear (caterpillar)	1,8	Planers	1,4	Briquetting presses	2,5
Travelling gear (rails)	1,6	Frame saws	1,4	Eccentric presses	2,0
Suction pumps	1,6	Steel plants		Forging presses	2,25
Bucket wheels	1,8	Blast furnace blowers	1,4	Brick moulding presses	2,5
Cutter heads	2,0	Converters	2,0	Pumps	
Slewing gear	1,4	Inclined blast furnace elevators	1,8	Centrifugal pumps (thin liquid)	1,25
Winches	1,6	Slag crushers	1,8	Centrifugal pumps (viscous liquid)	1,4
Mining, stones		Cranes		Reciprocating pumps ($U \leq 1:100$)	1,8
Crushers	2,24	Luffing gear	1,25	Reciprocating pumps ($U = 1:100-200$)	1,6
Rotary kilns	1,8	Traversing gear	1,6	Plunger pumps	2,0
Mine ventilators	2,0	Hoists	1,4	Sludgers	1,4
Vibrators	1,6	Slewing gear	1,4	ELMO-Vacuum pump	1,5
Chemical plant		Winches	1,25	Textile machines	
Agitators (thin liquid)	1,25	Metal working		Winders	1,6
Agitators (viscous liquid)	1,6	Press brakes	1,6	Printing and drying machines	1,6
Centrifuges (light)	1,4	Sheet straighteners	1,8	Tanning vats	1,6
Centrifuges (heavy)	1,8	Hammers	1,8	Calenders	1,6
Conveyor plants		Shears	1,6	Opening machines	1,6
Conveyors	1,8	Forging presses	1,8	Weaving looms	1,6
Slatted conveyors	1,6	Stamping machines	1,8	Compressors	
Belt conveyors (bulk materials)	1,4	Mills		Reciprocating compressors	2,0
Slatted conveyors (piece goods)	1,6	Hammer mills	2,0	Reciprocating compressors	1,6
Pocket belt conveyors	1,25	Ball mills	2,0	Turbo compressors	1,6
Bucket chain conveyors	1,4	Suspended roller mills	2,0	Rolling mills	
Rotary conveyors	1,4	Impact mills	2,0	Sheet metal shears	1,8
Elevators	1,4	Rod mills	2,0	Sheet turning machines	1,6
Bucket type flour elevators	1,25	Roller mills	2,0	Ingot slab mills	2,0
Lifts	1,8	Food machinery		Block conveyors	1,8
Apron conveyors	1,4	Fillers	1,25	Block pushers	2,0
Screw conveyors	1,4	Kneading machines	1,4	Tape and wire reels	1,4
Steel belt conveyors	1,4	Packaging machines	1,25	Descalers	1,6
Redler conveyors	1,4	Sugar cane crushers	1,6	Sheet mills	1,8
Blowers, ventilators		Sugar cane cutters	1,6	Plate mills	2,0
Rotary piston blowers	1,4	Sugar cane mills	1,8	Cold rolling mills	2,0
Blowers (axial and radial)	1,25	Sugar beet cutters	1,6	Track-type tractors	1,6
Cooling tower ventilators	1,4	Sugar beet washing plant	1,6	Billet shears	1,8
Induced draught fans	1,4	Paper machines		Cooling beds	1,4
Turbo-blowers	1,25	Couch presses	1,8	Transfer skids	1,4
Generators, converters		M. G. cylinders	2,0	Roller tables (light)	1,4
Frequency converters	2,24	Reels	1,8	Roller tables (heavy)	1,8
Generators	1,4	Beating engines	1,6	Roller levellers	1,6
Welding generators	2,24	Pulp grinders	1,8	Trimming shears	1,4
Rubber and plastics machinery		Calenders	1,6	End shears	1,8
Extruders	1,6	Wet presses	1,8	Looplifters	1,4
Calenders	1,6	Opening machines	1,8	Roller control gear	1,4
Kneader machines	1,8	Agitators	1,8		
Mixers	1,8	Suction presses	1,6		
Rolling plant	1,8	Suction couch rolls	1,8		
		Drying cylinders	2,0		

1) The service factor K_1 is applicable to electric motor or turbine drives.

For drives by hydraulic motor or internal combustion engines, the factor K_1 has to be multiplied by 1.1.

Size determination

Condition for continuous output

$$\frac{P_N}{n} \cdot K_1 \leq \frac{P_{KN}}{n} \text{ (kW} \cdot \text{min)}$$

P_N = max. contin. output (kW)

n = Operating speed (rpm)

K_1 = Service factor according to table 1

$\frac{P_{KN}}{n}$ = Power factor in accordance with dim. table

LC = Load Cycles

Condition for continuous torque T_N

$$T_N \cdot K_1 \leq T_{KN}$$

$$T_N = \frac{P_{KN}}{n} \cdot 9550 \text{ (Nm)}$$

$$T_{KN} = \frac{P_{KN}}{n} \cdot 9550 \text{ (Nm)}$$

Permissible additional loads:

max. starting load of coupling = $1.5 \cdot P_{KN}/n$ for 10^5 LC

max. short-circuit load of coupling = $3 \cdot P_{KN}/n$ for 10^3 LC

If higher angular or radial misalignments occur in operation, then it may be necessary to reduce the max. permissible operating speed. In this case, the speed factors in accordance with Table 2 must be used.

A further criterion when defining the coupling size is the max. permissible bore diameter. For this reason, the bore must be checked after output-related determination of the coupling. If this bore does not permit mounting of the existing shaft, then a correspondingly larger coupling must be selected.

Important note

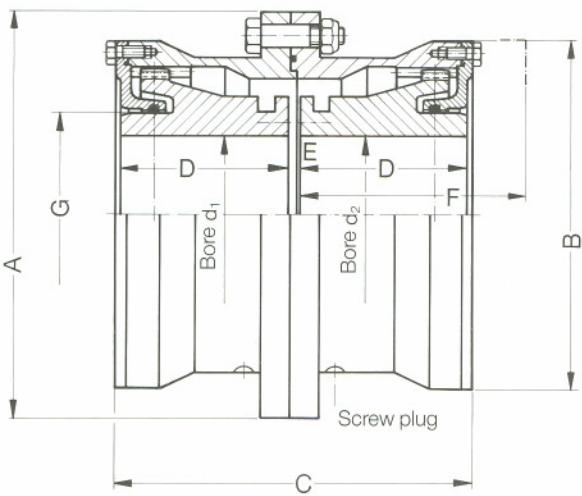
The values specified in the dimension tables for the max. permissible bore apply to keyed connections only if the groove height does not exceed the dimensions in accordance with DIN 6885. Please enquire if the groove is higher.

The power transmission capacity of the shaft-hub connection has to be verified by the purchaser.

Curved Tooth Couplings

Series SB

Dimension Table No. 243 125



The tables on page 33 contain information on torsional spring rates.

Dismounting dimension F is required for vertical installation and removal of machines.

The couplings of the SB series are also available in electrically insulated design for use in electrical installations.

Sizes 30 to 200 available ex stock.

Larger sizes on request.
Subject to dimensional modifications.

- 1) Referred to a permissible angular misalignment of $\Delta K_w \text{ perm.} = 1.5^\circ$ per coupling half. Higher values are possible but require special measures.
- 2) Values of the complete coupling with bore $d_1; d_{2 \text{ max.}}$.

Coupling type SB Size	Norm. cont. duty P_{KN} kW · min	Speed n _{max.} rpm	Dimensions									Max. static 1) radial misalign- ment $\Delta K_{w \text{ max.}}$ mm	Total grease quantity kg	Total oil quantity Liter	Mass moment of inertia 2) kgm ²	Weight kg	
			Bore $d_1; d_2$ pre mm	d_1 min mm	d_1 max mm	A mm	B mm	C mm	D mm	E mm	F mm						
30	0,082	7500	10	12	32	118	92	108	50	5	75	45	$\pm 1,95$	0,085	0,027	0,0064	4,4
40	0,146	6900	20	22	46	145	115	128	60	5	90	60	$\pm 2,70$	0,09	0,043	0,0170	7,5
50	0,288	6300	20	22	58	165	135	148	70	5	110	75	$\pm 3,00$	0,17	0,070	0,033	11,2
60	0,50	5900	26	28	70	200	160	172	80	6	120	90	$\pm 3,45$	0,25	0,11	0,082	18,4
70	0,82	5400	26	28	78	220	178	192	90	6	130	100	$\pm 3,90$	0,35	0,15	0,133	26
80	1,14	5000	30	32	92	240	196	212	100	6	150	120	$\pm 4,35$	0,40	0,20	0,200	32
90	1,64	4700	30	32	100	270	225	236	110	8	170	130	$\pm 4,80$	0,60	0,30	0,38	47
100	2,30	4300	53	55	110	280	240	256	120	8	180	140	$\pm 5,25$	0,75	0,35	0,49	54
110	2,88	4000	63	65	120	310	265	276	130	8	190	155	$\pm 5,70$	1,00	0,45	0,82	72
125	4,60	3700	73	75	138	340	295	320	150	10	215	175	$\pm 6,45$	1,3	0,65	1,35	100
140	6,48	3400	83	85	156	390	325	350	165	10	230	200	$\pm 7,20$	1,6	0,85	2,41	142
160	9,24	3100	118	120	180	435	370	404	190	12	270	230	$\pm 8,40$	2,6	1,4	4,3	199
180	12,92	2900	138	140	200	480	415	456	220	12	300	260	$\pm 9,60$	3,3	1,8	7,5	285
200	18,4	2700	158	160	225	545	465	512	245	14	340	290	$\pm 10,80$	4,8	2,5	14,1	420
220	25,6	2400	158	160	250	580	510	556	270	16	360	355	$\pm 12,00$	5	2,5	19,7	514
240	32,8	2200	178	180	275	645	560	598	290	18	380	390	$\pm 12,80$	7	3,5	29,9	657
260	41,0	2100	198	200	295	680	595	640	310	20	400	415	$\pm 13,50$	8	4	42,3	797
280	51,4	2000	218	220	325	745	660	702	340	22	440	460	$\pm 14,25$	10	6	69	1065
300	64,8	1900	238	240	340	775	675	744	360	24	470	480	$\pm 15,00$	11	8	84	1220
320	82,0	1800	258	260	375	825	725	786	380	26	500	525	$\pm 16,50$	13	9	119	1470
340	98,6	1700	278	280	400	915	795	808	390	28	520	560	$\pm 16,50$	20	11	184	1870
360	118,3	950	297	300	420	960	840	830	400	30	540	590	$\pm 16,50$	26	12	244	2245
380	138,0	900	317	320	445	1010	890	870	420	30	560	625	$\pm 18,00$	29	13	345	2610
400	165,1	875	337	340	460	1050	925	890	430	30	580	650	$\pm 18,00$	32	15	368	2780

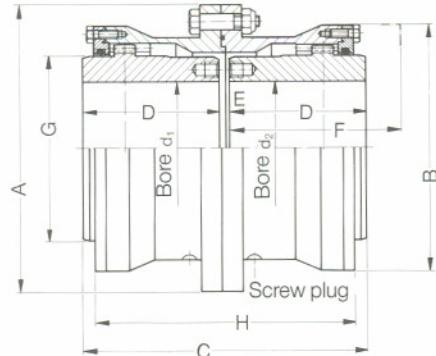
Curved Tooth Couplings

Series SBk and LBk

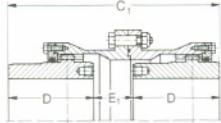


Dimension Table No. 243 126

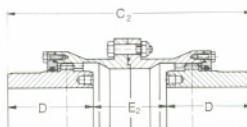
Series SBk



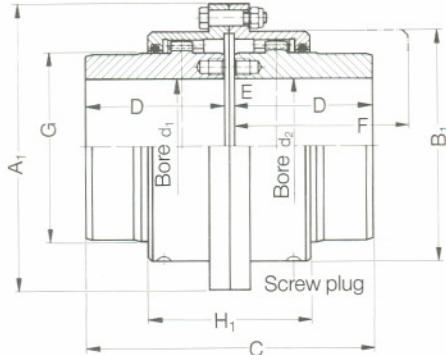
Arrangement II



Arrangement III



Series LBk



Sizes 32 to 225 available ex stock

Larger sizes on request.
Subject to dimensional modifications.

Dismounting dimension F is required for vertical installation and removal of machines.

The design of the SBk series permits different arrangements of the hubs in the housing, thus permitting even larger shaft distances to be bridged. In the case of tapered bores, the increased E-dimension offers space for installation of shaft nuts.

Both the SBk and LBk couplings are available in electrically insulated design for use in electrical installations.

The tables on page 33 contain information on the torsional spring rates.

- 1) Referred to a permissible angular misalignment of $\Delta K_{w\ perm.} = 0.75^\circ$ per coupling half. Higher values are possible but require special measures.
- 2) Values of the complete coupling for bores $d_1; d_{2\ max.}$

Size	Max. static ¹⁾ radial misalign- ment $\Delta K_{w\ max.}$ mm	Series LBk					
		Total grease quantity kg	Mass moment of inertia $J_{z\ z}$ kgm ²	Weight ²⁾ kg			
32	± 0,57	0,03	0,0034	2,9			
38	± 0,69	0,04	0,0059	4,3			
48	± 0,71	0,06	0,015	7,0			
60	± 0,77	0,10	0,026	9,3			
70	± 0,78	0,15	0,059	14,7			
80	± 0,84	0,22	0,097	20,0			
90	± 0,92	0,29	0,14	25,4			
100	± 1,08	0,44	0,28	38,0			
110	± 1,23	0,55	0,36	45,6			
125	± 1,34	0,79	0,64	62			
140	± 1,44	0,90	1,03	82			
160	± 1,70	1,23	1,5	120			
180	± 1,89	1,9	3,6	177			
200	± 2,12	2,4	6,2	245			
225	± 2,42	3,7	11,2	347			

Curved Tooth Couplings

Series SBD

Dimension Table No. 243137

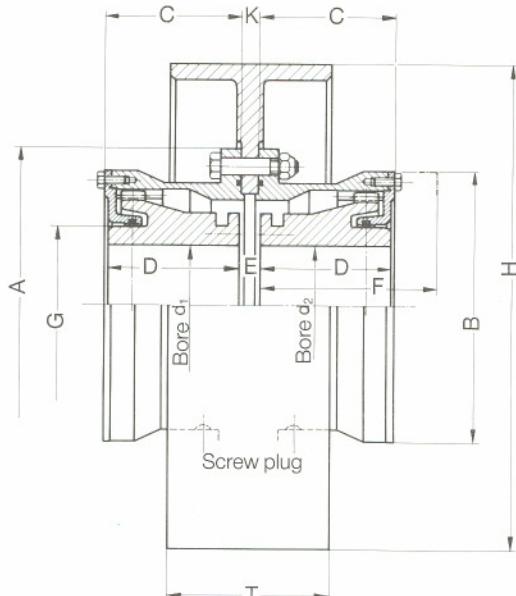
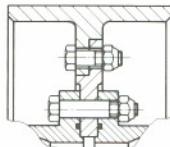


Fig. 2



Dismounting dimension F is required for vertical installation and removal of machines.

Split brake disks permit vertical installation and removal of machines.

1) Speed n_{\max} , depends on the permissible circumferential speed of the brake disk. Observe the specifications of the brake manufacturer. Calculate the circumferential speed in accordance with the formula on page 42.

2) Referred to a permissible angular misalignment of $\Delta K_{w,\text{perm.}} = 1.5^\circ$ per coupling half. These values apply only to the coupling and not to the braking device.

3) Values of the complete coupling without brake disk for bore $d_1; d_{2,\max}$.

The tables on page 33 contain information on the torsional spring rates.

Larger sizes on request.
Subject to dimensional modifications.

Recommended assignment of brake disks			
Coupl. size	Brake disk Ø H mm mm mm		
30	200	250	
40	200	250	315
50	200	250	315
60	250	315	400
70	250	315	400
80	315	400	
90	315	400	500
100	315	400	500
110	400	500	630
125	400	500	630
140	500	630	710
160	500	630	710
180	630	710	
200	630	710	

Brake disk dimension			Mass moment of inertia kgm ²	Weight kg
Ø H mm	T mm	K mm		
200	75	8	0,033	4,22
250	95	9	0,09	7,25
315	118	11	0,28	13,5
400	150	14	0,90	28
500	190	18	2,35	45
630	236	22	7,50	94
710	265	22	12,5	123

Curved Tooth Couplings

Series LBkD

Dimension Table No. 243141

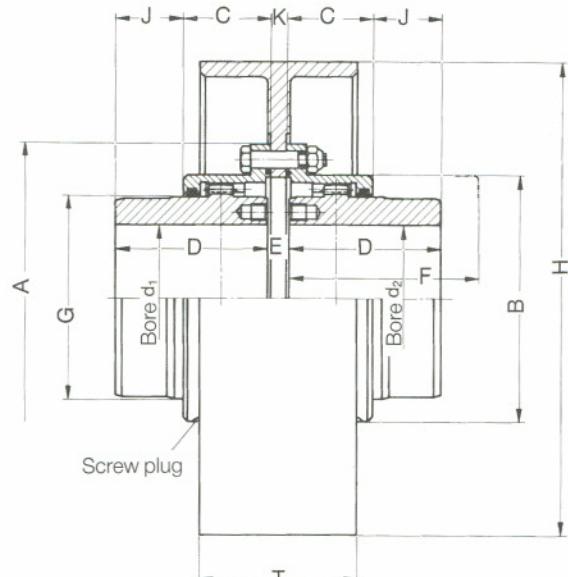
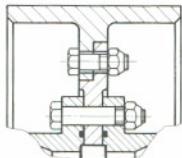


Fig. 2



Split brake disks permit vertical installation and removal of machines.

1) Speed n_{\max} depends on the permissible circumferential speed of the brake disk. Observe the specifications of the brake manufacturer. Calculate the circumferential speed in accordance with the formula on page 42.

2) Referred to a permissible angular misalignment of $\Delta K_{w,\text{perm.}} = 0.75^\circ$ per coupling half. These values apply only to the coupling and not to the braking device.

3) Values of the complete coupling without brake disk for bore d_1 ; $d_2 \max$.

The tables on page 33 contain information on the torsional spring rates.

Larger sizes on request.
Subject to dimensional modifications.

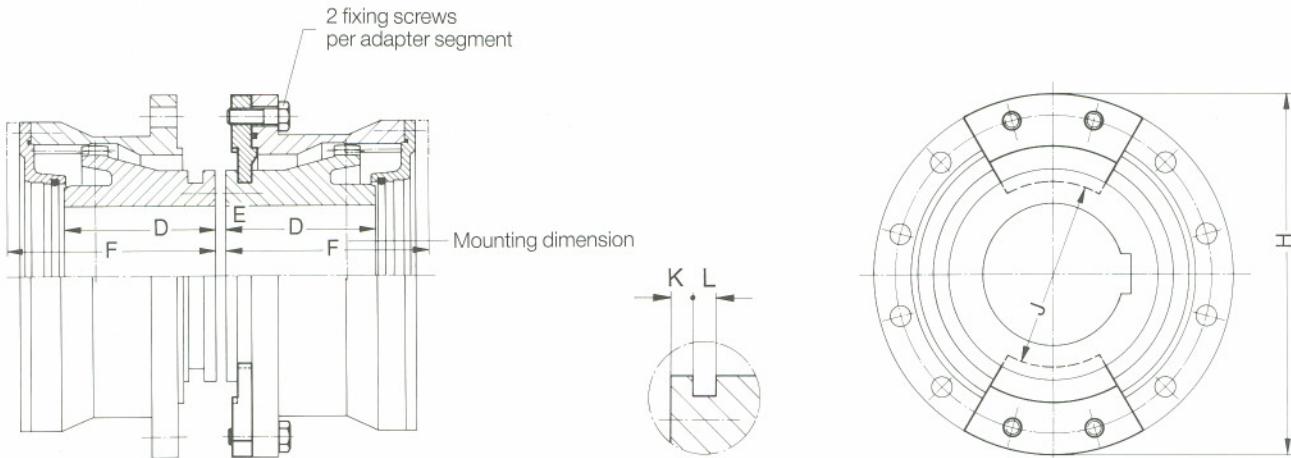
Recommended assignment of brake disk			
Coupling size	Brake disk Ø H mm	mm	mm
32	200		
38	200	250	
48	200	250	315
60	200	250	315
70	250	315	400
80	315	400	
90	315	400	
100	400	500	
110	400	500	
125	400	500	630
140	400	500	630
160	500	630	710
180	500	630	710
200	630	710	
225	710		

Brake disk dimension			Mass moment of inertia kgm ²	Weight kg
Ø H mm	T mm	K mm		
200	75	8	0,033	4,22
250	95	9	0,09	7,25
315	118	11	0,28	13,5
400	150	14	0,90	28
500	190	18	2,35	45
630	236	22	7,50	94
710	265	22	12,5	123

Adapter Segments for Curved Tooth Couplings

Series SB

Dimension Table No. 243146



Adapter segments for radial and axial guidance of one coupling housing when coupling halves are separated. It is thus possible to accelerate drive machines to the required speed without coupled work machine.

This is necessary, for example, when testing electric motors. Adapter segments are intended only for brief use and are not suitable for continuous operation.

The space corresponding to dimension F is required for installation and removal of the adapter segments.

Larger sizes on request.
Subject to dimensional modifications.

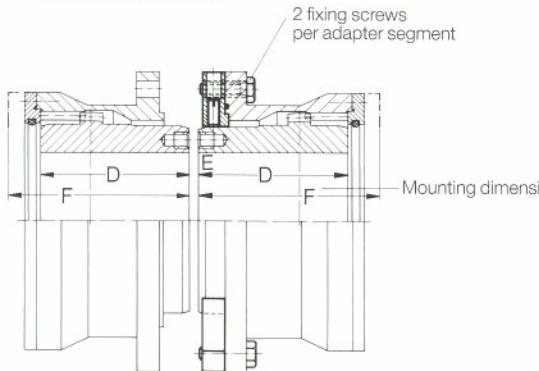
Coupling type SB Size	Dimensions							Dimensions of the pull-off threads in the hubs			
	D mm	E mm	F mm	H mm	J mm	K mm	L mm	Size of fixing screws	Thread diameter	Hole distance mm	Number of holes
30	50	5	75	115	48	4,5	5	M 6 x 20	—	—	—
40	60	5	90	145	62	4,5	5	M 8 x 20	—	—	—
50	70	5	110	165	75	4,5	6	M 8 x 25	—	—	—
60	80	6	120	195	85	7,0	6	M 10 x 25	—	—	—
70	90	6	130	215	105	8,0	8	M 10 x 30	M 8	94	2
80	100	6	150	230	118	8,5	8	M 10 x 30	M 8	107	2
90	110	8	170	265	128	8,5	10	M 12 x 35	M 10	115	2
100	120	8	180	270	145	10,5	10	M 12 x 35	M 12	130	2
110	130	8	190	305	156	11,0	12	M 16 x 40	M 12	140	2
125	150	10	215	330	180	11,0	12	M 16 x 45	M 16	162	2
140	165	10	230	375	210	12,0	13	M 18 x 45	M 16	190	2
160	190	12	270	425	240	14,0	14	M 22 x 55	M 20	215	2
180	220	12	300	470	270	14,5	14	M 22 x 55	M 24	240	2
200	245	14	340	535	310	15,0	15	M 27 x 60	M 24	270	2
220	270	16	360	580	325	21,5	18	M 27 x 60	M 24	290	2
240	290	18	380	645	350	22,0	19	M 33 x 60	M 24	320	2
260	310	20	400	680	375	23,0	20	M 33 x 60	M 30	340	2
280	340	22	440	745	420	24,0	21	M 36 x 70	M 30	380	2
300	360	24	470	775	435	26,5	24	M 36 x 75	M 36	395	2
320	380	26	500	825	470	28,0	25	M 36 x 75	M 36	430	2

Adapter Segments for Curved Tooth Couplings Series SBk and LBk

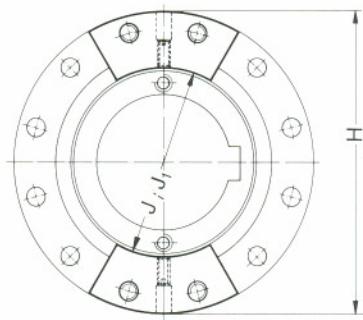
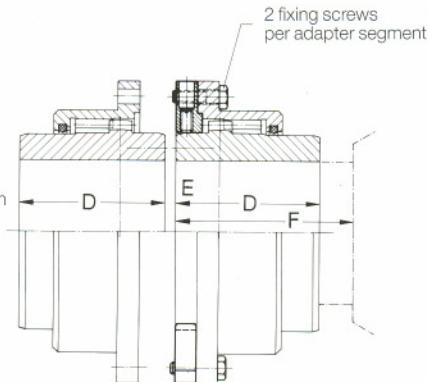


Dimension Table No. 243 147

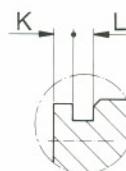
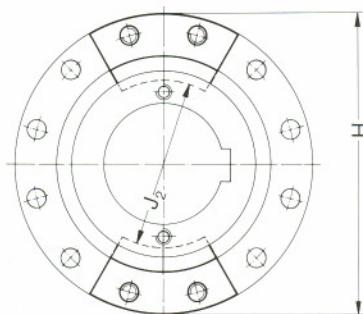
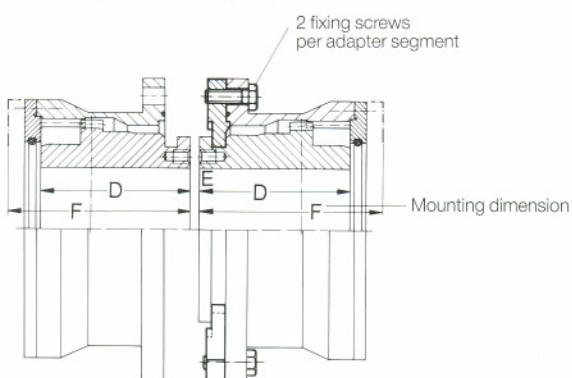
SBk couplings
with standard hub



LBk couplings



SBRk coupling with hub
design for retaining ring.



- 1) Dimension J applies to SBRk couplings with standard hub.
- 2) Dimension J₁ applies to LBk couplings.
- 3) Dimension J₂ applies to SBk couplings with hub design for retaining ring.

Larger sizes on request.
Subject to dimensional modifications.

Couplings- types SBk a. LBk	Dimensions								Size of fixing screws
	D	E	F	H	J ¹⁾	J ₁ ²⁾	J ₂ ³⁾	K	
Size	mm	mm	mm	mm	mm	mm	mm	mm	mm
32	50	4	80	105	—	48	—	—	M 6 x 20
38	60	5	90	115	60	60	54	4,5	M 6 x 20
48	70	5	100	145	76,2	77	69	4,5	M 8 x 20
60	80	6	110	165	89,2	90	81	4,5	M 8 x 25
70	90	6	120	195	111,5	112,5	98	7	M 10 x 25
80	100	6	130	215	127	128	113	8	M 10 x 30
90	110	8	140	230	144	145	128	8,5	M 10 x 30
100	125	8	150	265	158,5	160,5	138	8,5	M 12 x 35
110	140	8	170	270	174,6	176	152	10,5	M 12 x 35
125	150	10	180	305	199,6	200,5	171	11	M 16 x 40
140	170	10	200	330	222,9	224,5	195	11	M 16 x 45
160	190	12	230	375	254,9	256,5	225	12	M 18 x 45
180	220	12	260	425	286,9	288,5	255	14	M 22 x 55
200	250	14	300	470	318,8	320,5	290	14,5	M 22 x 55
225	280	16	330	530	359,8	362	332	15	M 27 x 60

Dimensions of the pull-off threads in the hubs		
Thread diameter	Hole distance mm	Number of holes
—	—	—
—	—	—
M 6	66	2
M 8	78	2
M 8	95	2
M 12	110	2
M 12	125	2
M 12	140	2
M 16	150	2
M 16	170	2
M 20	190	2
M 20	220	2
M 24	245	2
M 24	270	2
M 24	305	2

Radial Misalignment, Torsional Spring Rates for Couplings with Intermediate Shaft



In contrast to couplings with spacer, no minimum lengths are prescribed for intermediate shafts. The tables therefore only contain information about the coupling, but do take into account the parts of the intermediate shaft located inside the hub seats.

Coupling types SBG / SRG	Torsional spring rates	
Size	C_{T1}	C_{T2}
Nm/rad	Nm/rad	
30	$0,48 \cdot 10^6$	
40	$1,19 \cdot 10^6$	
50	$2,19 \cdot 10^6$	
60	$3,92 \cdot 10^6$	
70	$5,56 \cdot 10^6$	
80	$8,52 \cdot 10^6$	
90	$12,1 \cdot 10^6$	
100	$14,2 \cdot 10^6$	
110	$18,5 \cdot 10^6$	
125	$25,5 \cdot 10^6$	
140	$38,7 \cdot 10^6$	
160	$57,3 \cdot 10^6$	
180	$73,9 \cdot 10^6$	
200	$101,2 \cdot 10^6$	
220	$150,0 \cdot 10^6$	
240	$184,3 \cdot 10^6$	
260	$235,6 \cdot 10^6$	
280	$299,4 \cdot 10^6$	
300	$357,3 \cdot 10^6$	
320	$458,5 \cdot 10^6$	
340	$620,4 \cdot 10^6$	

Information required for the calculation of the torsional spring rates of curved-tooth couplings with intermediate shaft.

C_{T1} = Torsional spring rate of the coupling for bore d_3 ; $d_{4\max}$. This contains the values of the intermediate shaft with a diameter corresponding to d_1 ; $d_{2\max}$, but only in the area of the hub lengths D. The data must be calculated additionally for the free part of the shaft.

C_{T2} = Torsional spring rate of the intermediate shaft.

C_{T3} = Torsional spring rate of the complete coupling.

The following formula is used for calculation of the overall torsional spring rate.

$$C_{T3} = \frac{1}{\frac{1}{C_{T1}} + \frac{1}{C_{T2}}} \text{ (Nm/rad)}$$

Diameter d_1 ; d_2 normally = nominal size of the coupling. In the area of the hub length D, shaft diameters deviating from d_1 ; $d_{2\max}$, have only an insignificant influence on the overall torsional spring rate of the coupling.

Coupling types SBGk / SRGk	Torsional spring rates	
Size	C_{T1}	C_{T2}
Nm/rad	Nm/rad	
38	$1,02 \cdot 10^6$	
48	$2,08 \cdot 10^6$	
60	$3,40 \cdot 10^6$	
70	$6,30 \cdot 10^6$	
80	$9,15 \cdot 10^6$	
90	$12,0 \cdot 10^6$	
100	$17,5 \cdot 10^6$	
110	$20,0 \cdot 10^6$	
125	$27,1 \cdot 10^6$	
140	$36,7 \cdot 10^6$	
160	$54,9 \cdot 10^6$	
180	$72,8 \cdot 10^6$	
200	$96,8 \cdot 10^6$	
225	$131,9 \cdot 10^6$	
250	$180,7 \cdot 10^6$	
265	$218,2 \cdot 10^6$	
280	$275,1 \cdot 10^6$	
315	$347,9 \cdot 10^6$	
335	$415,6 \cdot 10^6$	
355	$528,1 \cdot 10^6$	
375	$705,6 \cdot 10^6$	

Coupling types LBGK / LRGK	Torsional spring rates	
Size	C_{T1}	C_{T2}
Nm/rad	Nm/rad	
32	$0,69 \cdot 10^6$	
38	$1,25 \cdot 10^6$	
48	$2,64 \cdot 10^6$	
60	$4,16 \cdot 10^6$	
70	$7,98 \cdot 10^6$	
80	$12,1 \cdot 10^6$	
90	$16,9 \cdot 10^6$	
100	$23,5 \cdot 10^6$	
110	$27,3 \cdot 10^6$	
125	$40,5 \cdot 10^6$	
140	$52,6 \cdot 10^6$	
160	$76,6 \cdot 10^6$	
180	$112,1 \cdot 10^6$	
200	$147,9 \cdot 10^6$	
225	$206,6 \cdot 10^6$	

The maximum permissible static radial misalignment depends on the permissible angular misalignment and on the distance between the tooth centers of both coupling halves. This distance is variable for couplings with spacer or intermediate shaft. The dimension tables thus do not contain any values for the permissible misalignment.

The following formula can be used for the calculation of these data:

- 1) The same formulae apply to couplings with built-in retaining ring as to the comparable design without retaining ring. However, the factor 0.010 must always be placed before the bracket.

Coupling with spacer, without retaining ring ¹⁾

C = Length of coupling half in accordance with type dimension list.

$L; L_1$ = Distance between flange surfaces of the spacer

$H; H_1$ = Length of the coupling housing in accordance with type dimension list.

Series SBL

$$\Delta_{Kr} = 0,026 \cdot (1,6 \cdot C + L)$$

Series SBLk

$$\Delta_{Kr} = 0,013 \cdot (1,6 \cdot H + L)$$

Series LBLk

$$\Delta_{Kr} = 0,013 \cdot (H_1 + L_1)$$

Couplings with intermediate shaft, without retaining ring ¹⁾

L = Overall length of the intermediate shaft, including hub seats.

D = Length of coupling hub in accordance with type dimension list.

Series SBG

$$\Delta_{Kr} = 0,026 \cdot (L - 1,6 \cdot D)$$

Series SBGk*

$$\Delta_{Kr} = 0,013 \cdot (L - 1,4 \cdot D)$$

Series LBGk

$$\Delta_{Kr} = 0,013 \cdot (L - 0,66 \cdot D)$$

* Applies only to couplings up to size 225.

Groove Seal Tooth Backlash Control Device

Sealing of feather key grooves

In the case of shaft-hub connections by means of feather keys, the inner hub sides of the curved tooth couplings must be sealed against oil or grease leaks. This applies particularly to couplings in retaining ring design which have a higher lubricant level than the basic type. Suitable media for sealing are lead, synthetic resin or other oil-resistant and grease-resistant sealing compounds.

The seal may be produced as shown below. A design in accordance with Fig. 1 or 2 is advantageous for non-continuous feather keys. A run-out for

the feather key groove is created by the recess in the hub. The sealing surface between the hub and shaft is thus preserved.

This sealing surface is not present for continuous feather keys. Here, we recommend a solution with sealing cover in accordance with Fig. 3 or 4. Please specify the required design when making an enquiry or ordering.

Tooth backlash control

The tooth backlash control device for curved tooth couplings permits simple and rapid checking of the coupling toothing at the place of operation. It is not necessary to dismantle the coupling for this.

The clearance between the internal and external tooth gearing is measured directly at the tooth contact. Any changes can be established by a direct comparison of the measured values with the given data. It is thus possible to assess the current condition of the coupling at any time and without difficulties. Regular checks performed in this way ensure that the coupling is always ready for operation.

To perform the measurement, it is necessary to remove the screw plug. The tooth flanks must then be moved so that they contact on one side by turning the outer sleeve. The tooth clearance between the free tooth flanks to the curved tooth center can now be measured. It is expedient to use a thickness gauge for this procedure. If the tooth flanks do not contact on one side, the overall clearance is obtained by adding the flank clearance on both sides.

The following series in this catalog can be equipped with the tooth backlash control device.

SB, SBK
SBR, SBRK
SBL, SBLK
SRL, SRLK
SBG, SBGK
SRG, SRGK
SBD, SBDK
SBT, SBT
VSB

Provision of this facility is possible for the comparable types of the LBk series and HBk couplings, but is considerably more expensive.

Fig. 1

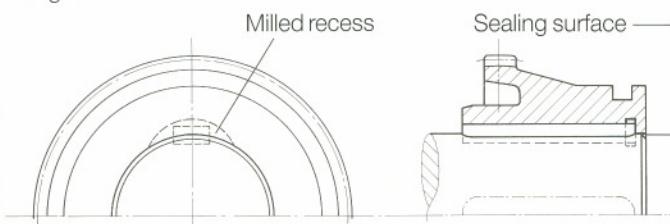


Fig. 2

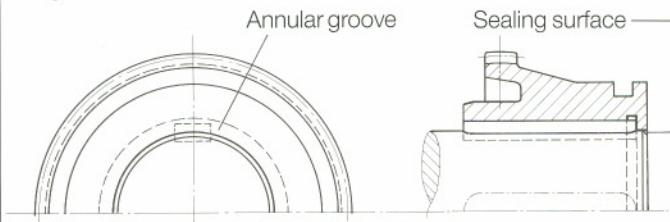


Fig. 3

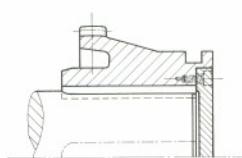
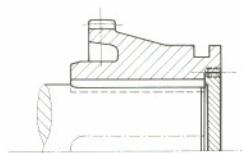
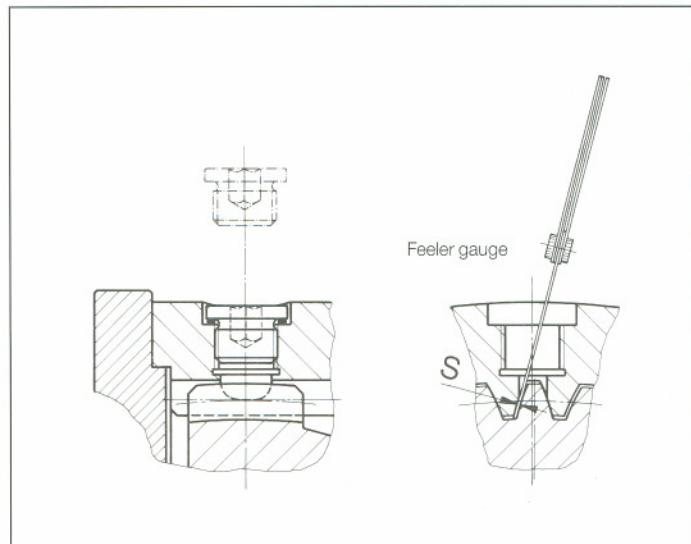


Fig. 4

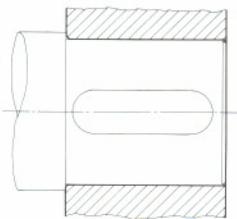


The Control device consists of an inspection opening in the coupling sleeve which is closed off by a screw with seal during operation. It is located over the tooth center of the curved-tooth gearing. An optional arrangement of several of these openings distributed around the coupling circumference is also possible.



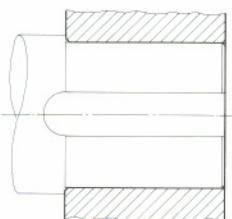
Shaft-Hub Connections

Fig. 1



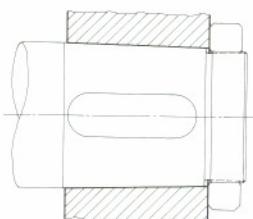
cyl. fit with 2 keys
and press fit

Fig. 2



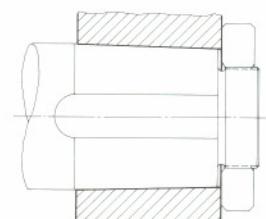
cyl. fit with 2 keys
and press fit

Fig. 3



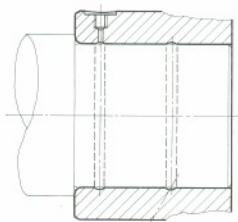
conical fit with 2 keys
and press fit

Fig. 4



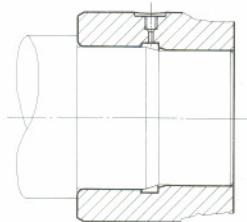
conical fit with 2 keys
and press fit

Fig. 5



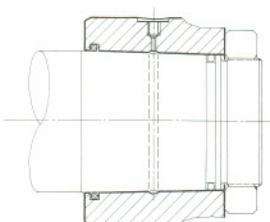
cyl. hydraulic fit

Fig. 6



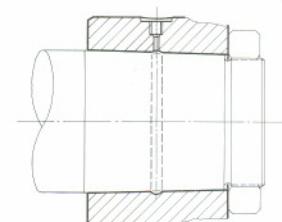
cyl. hydraulic fit
with stepped-down seat

Fig. 7



tapered hydraulic fit
with sealing rings

Fig. 8



tapered hydraulic fit

favourable

neutral

less favourable

Figure	1	2	3	4	5	6	7	8
Constructional utilisation of the given space								
Production Insensitiveness to damage								
Running quality								
axial positioning								
power transfer								
mounting warm								
mounting cold								
disassembly								

Curved Tooth Couplings Special Design

Fig. 1

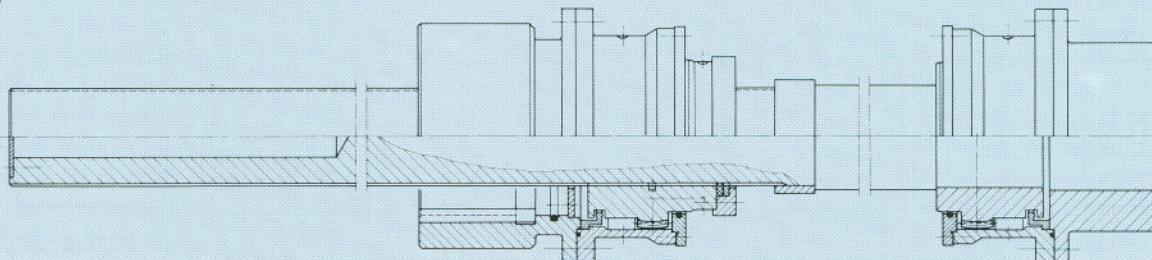


Fig. 2

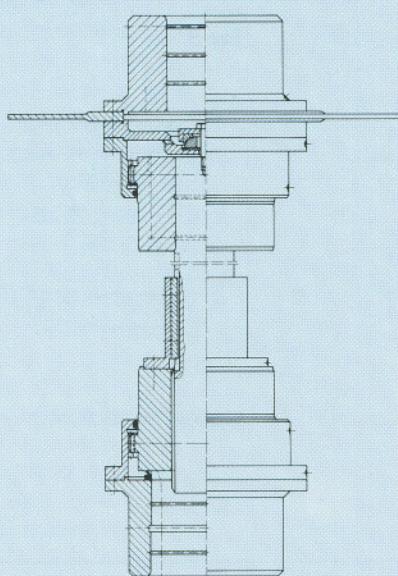


Fig. 3

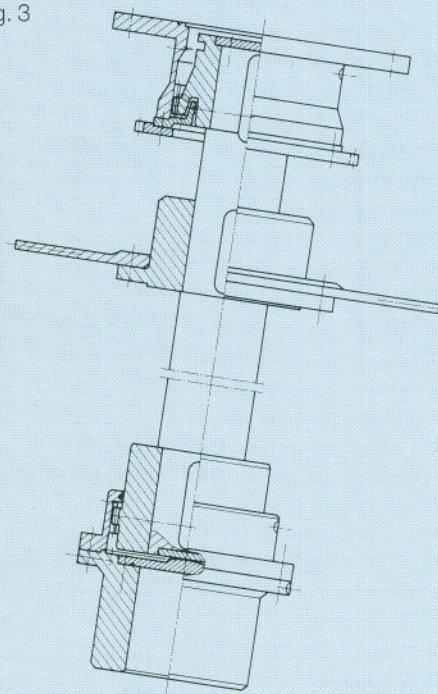
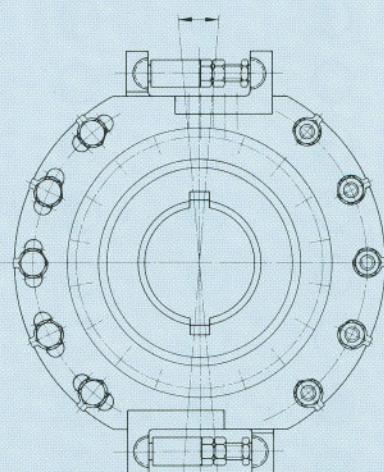
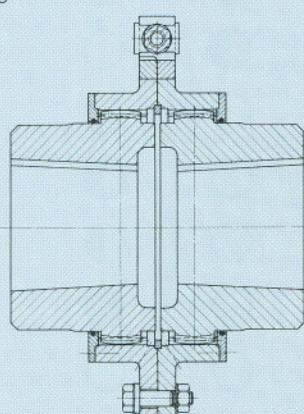


Fig. 4



1 Curved tooth coupling for driving a double winder.

2 Curved tooth coupling in vertical design and with built-on brake disk for operation in rudder propeller systems.

3 Curved tooth coupling in vertical design installed between water turbine and generator. The axis of rotation is tilted from the vertical by approx. 9°.

4 Curved tooth coupling with adjustment facility for driving a shear.

Fig. 5

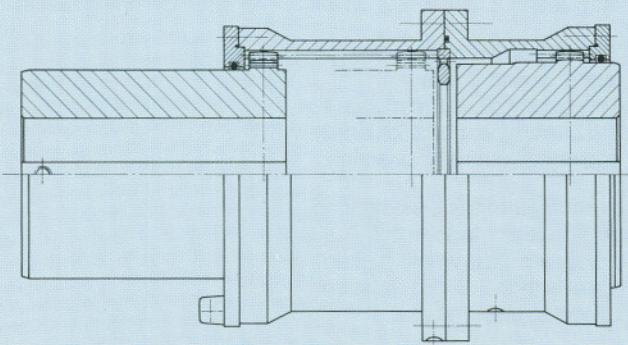
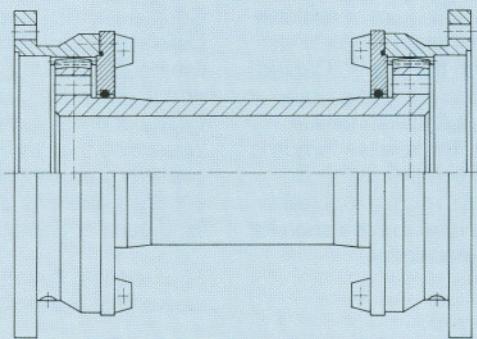


Fig. 6



- 5 Curved tooth coupling as refiner drive, with extended sleeve to allow axial displacement.
- 6 Curved tooth coupling with hub sleeve and flanged-on housings, used in ship drives.
- 7 Curved tooth coupling set up to hold a torque measuring shaft.
- 8 Curved tooth coupling in vertical design, electrically insulated, for driving a pump.
- 9 Curved tooth coupling with diaphragm seal for railway drives.

Fig. 7

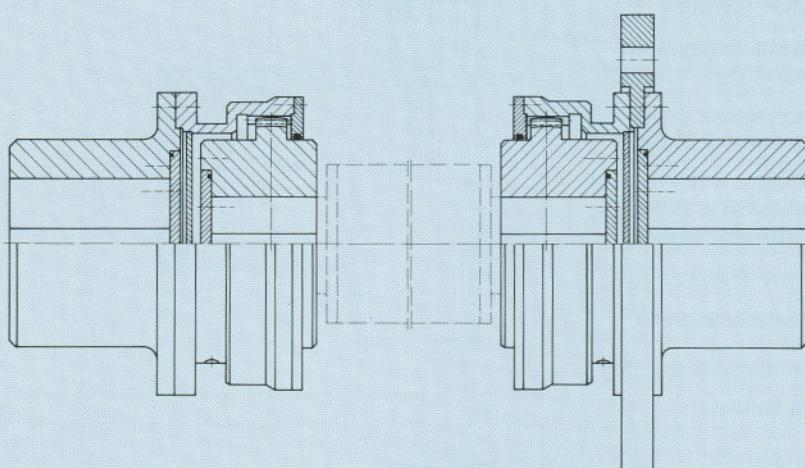


Fig. 8

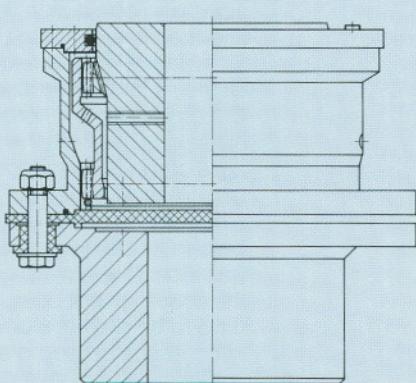
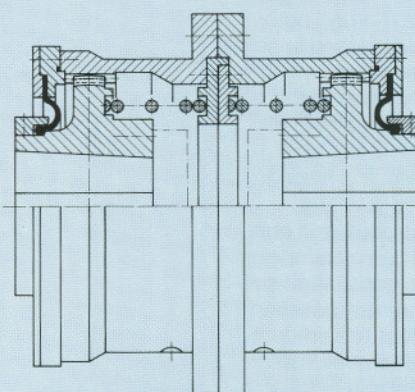


Fig. 9



HYGUARD® Safety couplings with Torque Limitation TORLOCK® Clamping Elements

Renk HYGUARD® Safety Couplings

ensure a maximum of reliability. The combination of safety elements with the proven Curved Tooth Coupling works like a safety coupling with torque limitation. The release torque is adjustable and remains constant over the entire operation time. The reliable protection for valuable machinery.

Advantages of the Renk HYGUARD® Safety Couplings:

- Transmission of the high torques within little space
- Torsionally rigid-flexible connection of shafts
- Overload protection with adjustable torque limitation
- Constant release torque over the entire duration of operation
- No interruption of operation due to material fatigue
- Low down-time costs
- Quick re-commisioning
- No cost-intensive stocking of spare parts

Operation

Basically, the safety element integrated into the coupling, consist of two bushes being inserted into each other, both ends of which are closed in pressure-tight manner. The little gap between both parts is filled with hydraulic oil through the valve bore, and then pressurized. This leads to expansion of the diameter. Built-in between shaft and hub this system acts as a clamping bush. Torque is transmitted by frictional engagement between the contact faces. The amount of transmittable torque is determined by oil pressure in the ring gap. This pressure is adjustable, and thus, the torque too.

Basic Types

The basic element of the Safety Coupling is the B type. Owing to its low dimensions, this coupling can be combined with an extremely large variety of drive elements without influencing their torque capacity. The coupling sleeve is equally suitable for clamping fixed parts and rotating bodies.

After release, the coupling sleeve acts like a plain bearing.

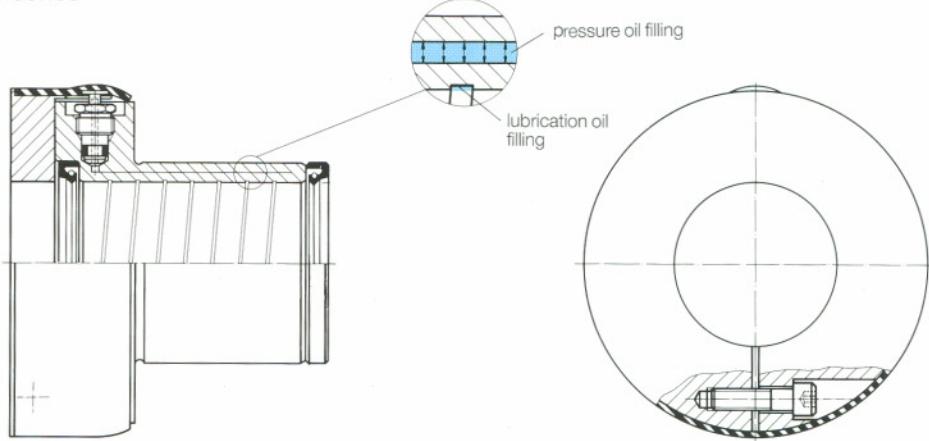
The circumferential speed at the sliding surface may be up to 1.5 m/s. The B series is preferably used for low-speed drives.

The BW series is designed for higher-speed drives. This essentially corresponds to the basic type, but additionally equipped with roller bearings.

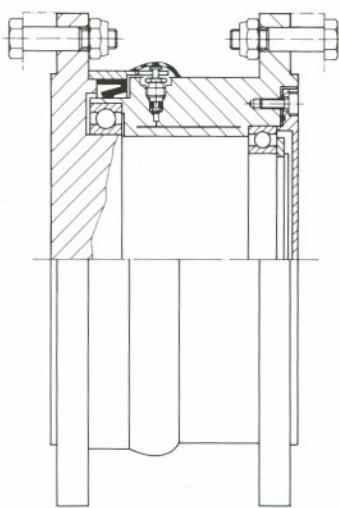
In contrast to the plain bearing version, this model is predominantly used where higher speeds than 1.5 m/s must be expected after release.

The space requirement is hardly larger than for the B version. No roller bearing is necessary for pure clamping tasks.

B series



BWL series



BWL series

Safety Couplings of the BWL series consist of a shaft part and a bush and thus already constitute a complete coupling unit. They are equipped on both sides with connection flanges and therefore can also be used, for instance as intermediate elements for Curved Tooth Couplings or other connection elements. This links the advantages of the Curved Tooth Couplings with those of the safety unit to obtain a flexible Safety Coupling with torque limitation. One advantage is that the coupling itself does not have to be altered. Conversions of couplings already in service are thus also possible.

Curved Tooth Couplings in combination with HYGUARD® Safety Elements

Protection owing to rapidness

is given by the safety disconnection, the essential components of which are the shear ring and the safety valve. There is a firm connection between shear ring and shaft, whereas the safety valve is accommodated in the movable part. If the operation torque exceeds the preset value, the safety element will slip on the profiled surfaces and the shear ring cuts off the top of the valve.

The system is depressurized within milliseconds and the connection is released.

Re-commissioning

The Renk Safety Coupling is easy to activate again. A high-pressure pump with pressure gauge for pressure setting and hydraulic oil will be sufficient to put into operation. The pump is connected to the injector, and with open valve the hydraulic oil is being pressed into the ring gap. If the pre-

determined pressure required for the torque transmission has been reached, the safety valve will be closed. The pressure chamber is now sealed hermetically, and the working pressure remains constant. The coupling is ready for use. Downtimes are thus reduced to a minimum. If the system has been disconnected due to overload, the safety valve is the only item to replace. Thereafter, the system is put into operation again as described above. The time-consuming replacement of shear pins or similar parts and

their storage are thus omitted. Stockkeeping of spare parts is limited to some valves, only.

Application Ranges

Due to the versatility of the Renk Safety Couplings and Clamping Elements, they are most suitable for many application areas, even under severest conditions. In rolling mills, e.g. where conventional couplings with shear pins or similar protection elements have previously been used, this safety system has been able to contribute to substantial cost savings.

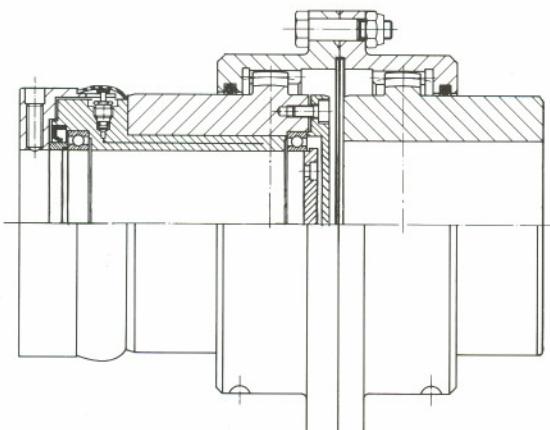
All machines which may be subjected to overload or lock-ups count to preferred ranges of application, e.g. in longitudinal shafts of paper mills, drafting equipment in the fibre industries, circular clamps of all kinds or solid pumps.

The Safety Coupling becomes controllable in combination with differential speed monitoring. This equipment releases an alarm, or disengages automatically, if the coupling has been released due to overload, leading to differing speeds between shaft and hub.

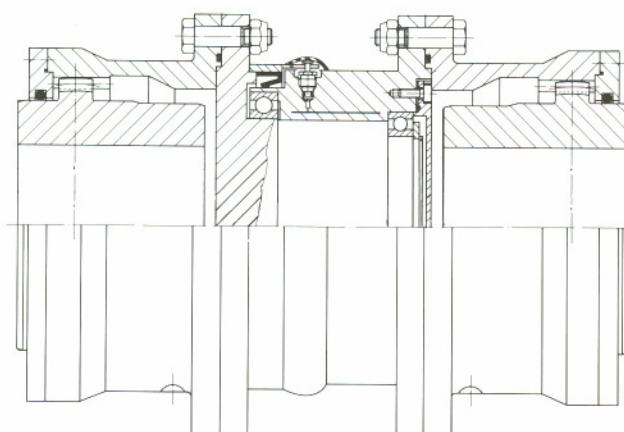
Clamping Elements TORLOC®

Wherever quick and safe tensioning of machinery parts is of vital importance, the Renk Safety Coupling as Clamping Element is offering itself. In test stands it is used as quick-grip coupling, and in machine tools and production machinery it is used as clamping element with fine adjustment possibility. The great advantages of this coupling when being used as quick-grip coupling are the easy handling as well as the fact that the coupling is put back into service again within shortest time. Both, manual release by opening the valve as well as controlled remote release is possible. For this purpose, special safety valves with extended shear tubes are available. In case of non-rotating clamped connections, remote release can be effected by means of high-pressure pipes and special valves.

BW-LBk combination



BWL-SBLk combination



Calculation Formulae

1. Critical Speed

Rough calculation of the critical whirling speed of intermediate shafts, spacers and hub sleeves.

$$n_k = 300 \cdot \sqrt{\frac{78,3 \cdot E \cdot I_{\text{axial}}}{G_1 \cdot l_0^4}}$$

n_k = critical whirling speed rpm

n = operating speed rpm

E = elastic modulus = $20,6 \cdot 10^4$ N/mm²

G_1 = weight for each 1 mm length in kg/mm (see item 5)

l_0 = tooth center distance mm

I_{axial} (see item 4)

permissible for subcritical operation

$$n \leq 0,75 \cdot n_k$$

for overcritical operation

$$n \geq 1,35 \cdot n_k$$

2. Torsional Spring Rate C_{T2}

$$C_{T2} = \frac{I_{\text{pol}} \cdot G}{l \cdot 1000}$$

C_{T2} = tors. spring rate of the intermed. shaft Nm/rad.

G = shear modulus = $7,95 \cdot 10^4$ N/mm²

l = length of the intermediate shaft in mm

l = $L - 2 \cdot D$

I_{pol} (see item 4)

3. Mass Moments of Inertia J

For the shaft to be applied:

$$J = \frac{G_2 \cdot d^2}{8 \cdot 10^6}$$

For the spacer (tube) to be applied:

$$J = \frac{G_2 \cdot (d_a^2 + d_i^2)}{8 \cdot 10^6}$$

J = mass moment of inertia kgm²

G_2 = weight kg (see item 5)

d = shaft diameter mm

d_a = outside diameter of spacer mm

d_i = inside diameter of spacer mm

4. Geometrical Moments of Inertia I

For the shaft to be applied:

$$I_{\text{axial}} = \frac{d^4}{20,37}$$

For the spacer (tube) to be applied:

$$I_{\text{axial}} = \frac{d_a^4 - d_i^4}{20,37}$$

I_{axial} = axial geometrical moment of inertia mm⁴

d = shaft diameter mm

d_a = outside diameter of spacer mm

d_i = inside diameter of spacer mm

I_{pol} = polar geometrical moment of inertia mm⁴

$$I_{\text{pol}} = 2 \cdot I_{\text{axial}}$$

5. Weight G_2

For the shaft to be applied:

$$G_2 = \frac{d^2 \cdot 6,165 \cdot l}{10^6}$$

For the spacer (tube) to be applied:

$$G_2 = \frac{(d_a^2 - d_i^2) \cdot 6,165 \cdot l}{10^6}$$

G_2 = weight kg

d = shaft diameter mm

d_a = outside diameter of spacer mm

d_i = inside diameter of spacer mm

l = length mm

G_1 = weight for each 1 mm length kg/mm

$$G_1 = \frac{G_2}{l}$$

6. Circumferential Speed

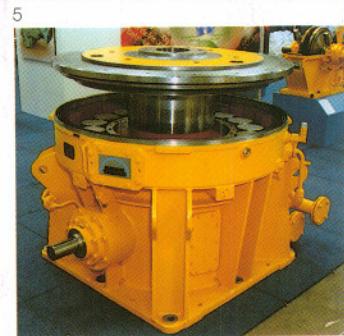
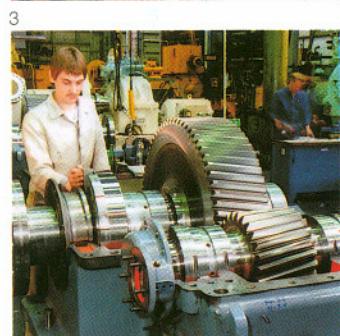
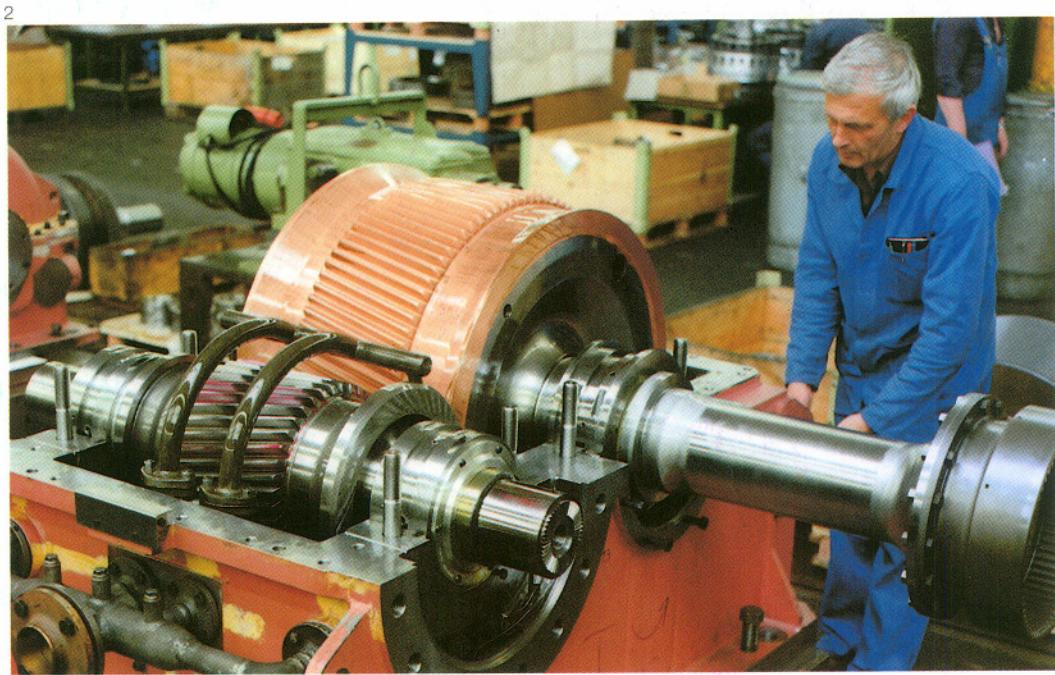
$$V = \frac{d \cdot n}{19100}$$

V = circumferential speed m/s

d = diameter of brake disk mm

n = operating speed rpm

A Few Examples from the Rest of our Product Range



1 Vertical curved tooth couplings with brake disk for use in rudder propeller systems.

2 Turbo gear for a transferable output of 24 000 kW at $n = 4460/14055$ rpm

3 Single-engine marine gear for a cargo vessel. Drive via diesel engine with 3530 kW at $n = 600/203$ rpm.

4 Injection-lubricated curved tooth coupling with torsion shaft from the Turbo range.

5 Vertical planetary gear in compact design for driving a roller mill in the cement industry.

6 Curved tooth couplings in special design with longitudinal compensation and roller-supported spacer.

RENK Aktiengesellschaft

Worldwide Your Partner with a Presence all over the World



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